

AMENDMENTS TO THE SPECIFICATION

Please replace paragraph [0013] beginning on page 6 with the following amended paragraph.

By combining springs having different spring constants, or providing a stopper which restricts the deflection amount of at least one of the plurality of springs arranged in series to below a preset value such that deflection deflection restriction by the stopper will develop while the input shaft is being pushed in to cause change in load increase of the spring relative to the brake operating amount, the load of the spring relative to the brake operating amount increases sharply from an intermediate point, so that relation between the brake operating amount and the output hydraulic pressure approaches an ideal curve, thus improving the brake feeling.

Please replace paragraph [0017] beginning on page 8 with the following amended paragraph.

The piston 6 has a large-diameter portion 6a at its tip, which is airtightly and axially slidably mounted on a cylindrical portion of the power plate 8. The piston 6 receives the pressures in the constant pressure chamber 3 and the variable pressure chamber 4 on its pressure receiving surfaces opposite to each other. When a differential pressure between them is produced, thrust is produced, so that the piston 6 advances leftwardly in the figure. On the front surface of the piston 6, a plurality of circumferentially arranged pins [[16]] 15 are provided which extend through the power plate 8 and protrude into the constant pressure chamber 3. At the tip of each

pin [[16]] 15, a retainer 17 is provided. The spring 7 is disposed between the retainer 17 and the inner wall of the fixed shell 5.